Numerical Modeling of a Turbine

Modeling of Non-Smooth Dynamics of lamellas Efficient solutions for ODEs with periodic boundary conditions

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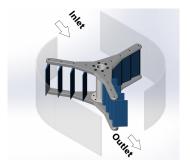


Background

- The turbine is called Oryon Watermill (OWM).
- Developed by Deep Water Energy BV, Netherlands.

Key features

- Modular build
- Operates under low pressure head conditions.
- 'Special' design of the rotor arm.



Numerical Model

• What are the expectations from the Numerical Model?

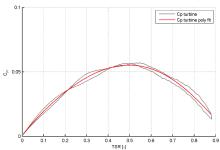


Numerical Model

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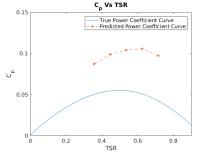
Performance Characteristic Curve

Co-efficient of Power v/s Tip Speed Ratio

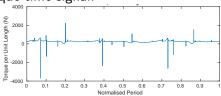


Current Numerical Model

- Computationally intensive
- No agreement with experimental result.

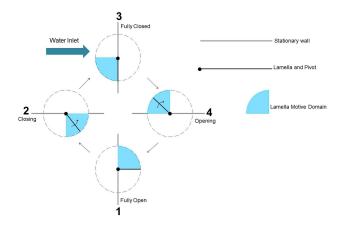


• Spikes in torque time signal.



Solution to the spikes in torques

- $\bullet \ \, \mathsf{Torque} \ \mathsf{transfer} \ \mathsf{mechanism-} \ \mathsf{Water} \to \mathsf{Lamella} \to \mathsf{Shaft}$
- Fluid-Structure Interaction problem.



Newton's Second Law $\label{eq:mathematically} \text{Mathematically expressed as an ODE-} \\ \mathcal{M} \ddot{\mathbf{x}}(t) = \mathbf{h}(t)$

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Non-smooth Dynamics

Position and velocity vectors are not smooth functions of time.

Numerical Solution



Numerical Solution

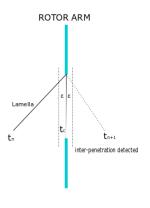
- Event Driven
 - Separate the non-smooth motion.
 - Integrate the smooth part until collision.
 - Solve the impact problem at the discontinuity.
 - Reset the ODE.

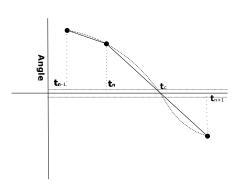
Numerical Solution

Event Driven

- Separate the non-smooth motion.
- Integrate the smooth part until collision.
- Solve the impact problem at the discontinuity.
- Reset the ODE.
- Time Stepping
 - Discretize the entire DAE with the inequalities.
 - Less administrative effort
 - Problems
 - Small time step size.
 - Poor accuracy as compared to the event-driven approach.
 - Inability to model the partial elastic behavior correctly.

Event Driven Approach





Event Driven Approach

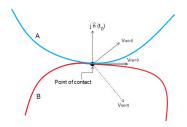
Constraints are formulated as contacts

- Colliding Contact
 Change in velocity on collision and bodies move apart with different velocities.
- Resting Contact
 Two bodies after collision are resting on each other.

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- v_{rel} > 0 → the bodies are not contacting after t_c.
- $v_{rel} < 0 \rightarrow$ The bodies are in colliding contact after t_c .
- $v_{rel} = 0 \rightarrow$ The bodies are in resting contact after t_c .

Colliding and Resting Contact

Colliding Contact

At the instant of collision-

- Calculate relative velocities.
- Add impulse $j = -M_a(1 + \epsilon)v_{rel}^-$
- Reset ODE.

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Resting contact

Contact force equals the force acting on the lamella exerted by the fluid.

ODE Formulation

- ODE Formulation
- Detection of time of collision

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- Detection of time of collision
- Detection of the type of contact
 - Colliding Contact
 - Resting Contact

Coupled system of nonlinear ODEs

Governing equation for lamella motion

$$\mathcal{M}\ddot{\mathbf{x}}(t) = \mathbf{h}(t) + \mathbf{w}(t); \tag{1}$$

Governing equation for fluid flow

$$\frac{d\mathbf{u}}{dt} + \mathbf{R}(\mathbf{u}) = 0; \tag{2}$$

- RHS of ODE 1 from forces due to fluids.
- RHS of ODE 2 from lamella motion.
- Resulting system of nonlinear ODEs has a periodic solution.

Requirements for the numerical method

- The method should be faster than the direct time integration.
- The method should have low memory requirements.
- The method should be easy to implement with minimum modifications to the solver.

Methods for solving the coupled system

Standard Methods for nonlinear ODEs with periodic solutions-

- Shooting Method
- Finite Difference Method
- Collocation Method

Specific Methods for fast analysis of periodic flows-

- Multitime multigrid method
- Time Linearization method
- Time spectral (harmonic balance) method

$$\frac{du}{dt} + R = 0; (3)$$

Fourier series expansion of u(t) with n harmonics reads:

$$u(t) = \sum_{i=0}^{n} u_{j} e^{ij\omega t}, \tag{4}$$

and the expansion for R(t) reads:

$$R(t) = \sum_{i=0}^{n} \mathfrak{R}_{j} e^{ij\omega t}, \tag{5}$$

where \mathfrak{u}_i and \mathfrak{R}_i are the Fourier co-efficients.

Inserting equations (4) and (5) in equation (3) we obtain,

$$\omega \sum_{j=0}^n ij\mathfrak{u}_j e^{ij\omega t} + \sum_{j=0}^n \mathfrak{R}_j e^{ij\omega t} = 0.$$

n equations for sine
$$\begin{cases} -1\omega\mathfrak{u}_{\mathsf{c}_1} + \mathfrak{R}_{\mathsf{s}_1} = 0; \\ -2\omega\mathfrak{u}_{\mathsf{c}_2} + \mathfrak{R}_{\mathsf{s}_2} = 0; \\ \vdots \\ -n\omega\mathfrak{u}_{\mathsf{c}_n} + \mathfrak{R}_{\mathsf{s}_n} = 0; \end{cases} \tag{6}$$

center-
$$\mathfrak{R}_0 = 0$$
 (7)

n equations for cosine
$$\begin{cases} 1\omega\mathfrak{u}_{\mathsf{s}_1}+\mathfrak{R}_{\mathsf{c}_1}=0;\\ 2\omega\mathfrak{u}_{\mathsf{s}_2}+\mathfrak{R}_{\mathsf{c}_2}=0;\\ &\vdots\\ \mathsf{n}\omega\mathfrak{u}_{\mathsf{s}_\mathsf{n}}+\mathfrak{R}_{\mathsf{c}_\mathsf{n}}=0; \end{cases} \tag{8}$$

$$\omega \mathcal{A}\mathfrak{u} + \mathfrak{R} = 0. \tag{9}$$

Frequency to time domain transformation-

$$\mathbf{u} = \mathcal{E}\hat{\mathbf{u}}(\mathsf{t}).$$

The operator \mathcal{E} is given by:

$$\omega \mathcal{A} \mathcal{E} \hat{\mathbf{u}} + \mathcal{E} \hat{\mathbf{R}} = 0.$$

Multiplying from left, the inverse transform operator (\mathcal{E}^{-1}) , we have:

$$\omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E})\hat{\mathbf{u}} + \hat{\mathbf{R}}(\hat{\mathbf{u}}) = 0. \tag{11}$$

- The derivative term is converted to a source term with the left multiplication of the operator $\mathcal{E}^{-1}\mathcal{A}\mathcal{E}$
- $\omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E})\hat{\mathbf{u}} + \hat{\mathbf{R}}(\hat{\mathbf{u}}) = 0$ can be solved as coupled stationary problems.

$$B_i = \sum_{k=1}^n ksin(k\omega jt_1); \quad j = \{1, ..., 2n\}.$$

Pseudo time marching method:

$$\begin{split} \frac{\mathrm{d}\hat{\mathbf{u}}}{\mathrm{d}\tau} + \omega \mathcal{B}\hat{\mathbf{u}} + \hat{\mathbf{R}}(\hat{\mathbf{u}}) &= 0; \\ \text{where} \quad \mathcal{B} &= \mathcal{E}^{-1}\mathcal{A}\mathcal{E}. \end{split} \tag{12}$$

$$\frac{\hat{\mathbf{u}}^{k+1} - \hat{\mathbf{u}}^k}{\Delta \tau} = -[\omega \mathcal{B}\hat{\mathbf{u}} + \hat{\mathbf{R}}(\hat{\mathbf{u}}^{k+1})]. \tag{13}$$

 $\hat{\mathbf{R}}(\hat{\mathbf{u}}^{k+1})$ is linearized using a Taylor Series expansion:

$$\hat{\mathbf{R}}(\hat{\mathbf{u}}^{k+1}) = \hat{\mathbf{R}}(\hat{\mathbf{u}}^{k}) + \mathcal{J}_{R}\Delta\hat{\mathbf{u}} + \mathcal{O}(\Delta\hat{\mathbf{u}}^{2}), \tag{14}$$

where \mathcal{J}_{R} is the Jacobian matrix of the residual vector in block diagonal form.

Explicit $\omega \mathcal{B}\hat{\mathbf{u}}$

$$\begin{bmatrix} \frac{\mathbf{V}\mathcal{I}}{\Delta\tau} + \mathcal{J}_{R} \end{bmatrix} \Delta \hat{\mathbf{u}} = -\hat{\mathbf{R}}^{k} - \omega \mathcal{B} \hat{\mathbf{u}}^{k}, \tag{15}$$

$$\begin{bmatrix} \mathsf{E}_{1} & 0 & \dots & 0 \\ 0 & \mathsf{E}_{2} & \ddots & \vdots \\ \vdots & & \ddots & 0 \\ 0 & \dots & 0 & \mathsf{E}_{2n+1} \end{bmatrix}; \quad \mathsf{E}_{i} = \frac{\mathsf{V}}{\Delta\tau_{i}} + \mathcal{J}_{\mathsf{ts}_{i,i}}$$

- ullet Solve independently for each of the 2n+1 stationary solutions
- Only the kth snapshot of the Jacobian has to be stored. No extra memory.
- Can be easily parallelized.
- Restricts the size of the Courant-Friedrich-Lewy (CFL) number and thus, time step size.

Implicit $\omega \mathcal{B}\hat{\mathbf{u}}$

$$\left[\frac{\mathsf{V}\mathcal{I}}{\Delta\tau} + \mathcal{J}_{\mathsf{R}} + \omega\mathcal{B}\right]\Delta\hat{\mathbf{u}} = -\hat{\mathbf{R}}^{\mathsf{k}} - \omega\mathcal{B}\hat{\mathbf{u}}^{\mathsf{k}};\tag{16}$$

$$\begin{bmatrix} \begin{smallmatrix} E_1 & H_{1,2} & H_{1,3} & \dots & H_{1,2n+1} \\ H_{2,1} & E_2 & \dots & \vdots \\ \vdots & & \ddots & & \vdots \\ H_{2n+1,1} & \dots & \dots & E_{2n} & H_{2n,2n+1} \\ & & & \vdots & & \vdots \\ \end{smallmatrix} \end{bmatrix}; \quad E_i = \frac{V}{\Delta \tau_i} + \mathcal{J}_{ts_{i,i}}; \quad H_{i,j} = V \omega \mathcal{B}_{i,j}$$

- Memory requirements and CPU time are more than the explicit approach.
- No restriction on CFL number and thus, on time step size.
- Significant modification in the solver.

Summary

- To solve the problem of unphysical spikes equations from the field of non-smooth dynamics to be used.
- The event-driven approach with the consideration of colliding and resting contacts is most appropriate.
- Complex problem of modeling lamella motion was reduced to simple ODE integration but with appropriate conditions.
- Coupled system of nonlinear ODEs with periodic solutions.
- The harmonic balance method is most suited for the current problem.

Research Questions

- How to apply the harmonic balance method to the coupled fluid-structure problem?
- Which of the two treatments-explicit or implicit, is the most appropriate?

Appendix

Methods Applied to a system of linear ODEs

Consider the a system of linear ODEs

$$\dot{\mathbf{x}}(t) = \mathcal{Q}\mathbf{x} + \mathbf{f}(t); \quad \mathbf{x}(0) = \mathbf{x}(\mathsf{T}); \quad \mathbf{x} = \begin{bmatrix} 0 & 1 \end{bmatrix}^\mathsf{T};$$
 (17)

where
$$\mathcal{Q} = \begin{bmatrix} 0 & -\omega \\ \omega & 0 \end{bmatrix}$$
 and $\mathbf{f}(\mathbf{t}) = \begin{bmatrix} \sin(\omega \mathbf{t}) \\ \cos(\omega \mathbf{t}) \end{bmatrix}$

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where
$$Q = \begin{bmatrix} 0 & -\omega \\ \omega & 0 \end{bmatrix}$$
 and $\mathbf{f}(\mathbf{t}) = \begin{bmatrix} \sin(\omega \mathbf{t}) \\ \cos(\omega \mathbf{t}) \end{bmatrix}$

The true solution of the above system is computed to be-

$$\begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} -\sin(\omega t) \\ \cos t(\omega t) + \frac{1}{\omega}(\sin(\omega t)) \end{bmatrix}$$
 (18)

Harmonic balance solution

$$\dot{\mathbf{x}}(t) = \mathcal{Q}\mathbf{x} + \mathbf{f}(t); \tag{19}$$

Applying harmonic balance method, gives:

$$\omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E})\hat{\mathcal{X}}+\hat{\mathcal{R}}=0.$$

where, for n harmonics we have

$$\mathcal{E}^{-1}\mathcal{A}\mathcal{E} \in \mathbb{R}^{2n+1\times 2n+1}; \quad \hat{\mathcal{X}} \in \mathbb{R}^{2n+1\times 2}; \quad \hat{\mathcal{R}}^{2n+1\times 2}$$

with

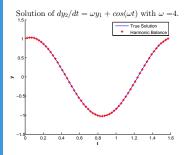
$$\hat{\mathcal{R}} = -\hat{\mathcal{X}}\mathcal{Q}^T - \mathcal{F}$$

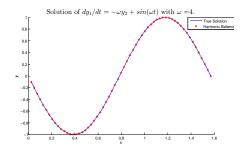
The final system looks like:

$$\omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E})\hat{\mathcal{X}} - \hat{\mathcal{X}}\mathcal{Q}^{\mathsf{T}} = \mathcal{F}$$

The above system is a Sylvester equation.

Plots







Another formulation

Sylvester problem can be represented as a linear equation in this case.

$$\omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E})\hat{\mathcal{X}} - \hat{\mathcal{X}}\mathcal{Q}^{\mathsf{T}} = \mathcal{F} \Rightarrow \mathcal{Z}\hat{\mathbf{x}} = \mathbf{F}$$
 (20)

where $\mathcal{Z} = \omega(\mathcal{E}^{-1}\mathcal{A}\mathcal{E}) - (\mathcal{I}\otimes\mathcal{Q})$

Observation

Comparing with $\mathcal{Z} = \mathcal{M} - \mathcal{N}$ splitting.

Properties of $\ensuremath{\mathcal{Z}}$ determine the convergence of the iterative process.

Van der Pol's equation

The Van der Pol's equation in reduced form:

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix}' = \begin{bmatrix} y_2 \\ \mu(1 - y_1^2)y_2 - y_1 \end{bmatrix} =: -\mathbf{R}$$
 (21)

Performing the harmonic balance transformation, we have:

$$\begin{split} \omega\mathcal{B}\hat{\mathbf{Y}} + \hat{\mathbf{R}}(\hat{\mathbf{Y}}) &= 0;\\ \text{where} \quad \hat{\mathbf{Y}} \in \mathbb{R}^{2*(2n+1)\times 1}; \quad \hat{\mathbf{R}} \in \mathbb{R}^{2*(2n+1)\times 1};\\ \mathcal{B} \in \mathbb{R}^{2*((2n+1)\times (2n+1))} \end{split} \tag{22}$$

Van der Pol's equation

Pseudo time stepping-

$$\frac{\mathrm{d}\hat{\mathbf{Y}}}{\mathrm{d}\tau} + \omega \mathcal{B}\hat{\mathbf{Y}} + \hat{\mathbf{R}} = 0. \tag{23}$$

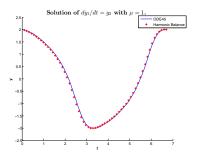
Explicit

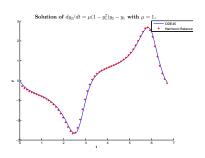
$$\left[\frac{\mathsf{V}\mathcal{I}}{\Delta\tau} + \mathcal{J}_{\mathsf{ts}}\right] \Delta \hat{\mathbf{Y}} = -\hat{\mathbf{R}}^{\mathsf{k}} - \omega \mathcal{B} \hat{\mathbf{Y}}^{\mathsf{k}}, \tag{24}$$

Implicit

$$\left[\frac{V\mathcal{I}}{\Delta\tau} + \mathcal{J}_{ts} + \mathcal{B}\right] \Delta \hat{\mathbf{Y}} = -\hat{\mathbf{R}}^{k} - \omega \mathcal{B} \hat{\mathbf{Y}}^{k}; \tag{25}$$

Plots





Though the Roads been rocky, it sure feels good to me

-Bob Marley